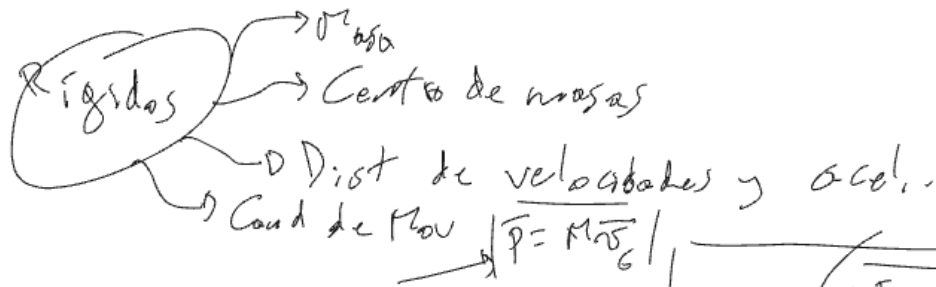


Proyecto 6: Dinámica del rígido en el plano

Repaso:



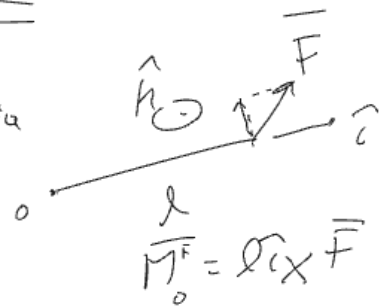
→ Ecuaciones cardinales: $M\vec{a}_G = \vec{R}^{EXT}$ (1ª Cardinal)

$\vec{L}_Q = M(\vec{r}_G - \vec{r}_Q) \times \vec{v}_Q + \mathbb{I}_Q \vec{\omega}$ En la misma base

↓

$\frac{d\vec{L}_Q}{dt} = M\vec{v}_G \times \vec{v}_Q + \vec{M}_Q^{EXT}$ (2ª Cardinal)

Tensor de inercia



$M(\vec{v}_G - \vec{v}_Q) \times \vec{v}_Q + \frac{d(\mathbb{I}_Q \vec{\omega})}{dt} = \vec{M}_Q^{EXT}$ (2ª Cardinal)

$T = \frac{1}{2} M v_p^2 + \frac{1}{2} \vec{\omega} \cdot \mathbb{I}_p \vec{\omega} + M \vec{v}_p \cdot [\vec{\omega} \times (\vec{r}_G - \vec{r}_p)]$

→ Base de $\bar{\omega}$ y de \mathbb{I} tiene que ser la misma!!
 ↑ Matriz 3x3 simétrica.

Potencia:
$$P = \overline{R}^{Ext} \cdot \bar{\omega}_Q + \overline{M}^{Ext} \cdot \bar{\omega}$$

Caso plano: $\uparrow \delta \theta \hat{n}$ $\bar{\omega} = \begin{pmatrix} 0 \\ 0 \\ \omega \end{pmatrix}$

$\mathbb{I}_Q^{plano} = \begin{pmatrix} \boxed{\text{NO ME IMPORTA}} & 0 \\ & 0 \\ 0 & 0 & I_{33} \end{pmatrix}$

$\mathbb{I}_Q \bar{\omega} = \begin{pmatrix} \boxed{\cdot} & 0 \\ & 0 \\ 0 & 0 & I_{33} \end{pmatrix} \begin{pmatrix} 0 \\ 0 \\ \omega \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ I_{33} \omega \end{pmatrix}$

Solo voy a calcular la estructura I_{33} .

En el plano tengo 3 ees y 3 incógnitas.

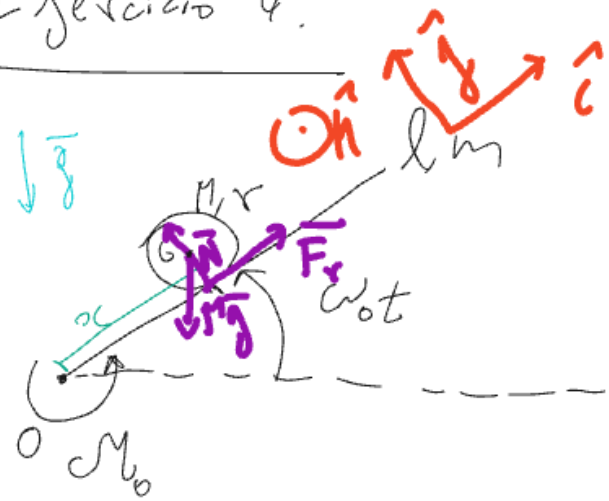
- Cambio de punto de aplicación de momentos:

$$\begin{cases} \overline{L}_{Q_1} = \overline{L}_{Q_2} + \overline{P} \times (Q_1 - Q_2) \\ \overline{M}_{Q_1} = \overline{M}_{Q_2} + \overline{R}^{Ext} \times (Q_1 - Q_2) \end{cases}$$



Ejercicio 4:

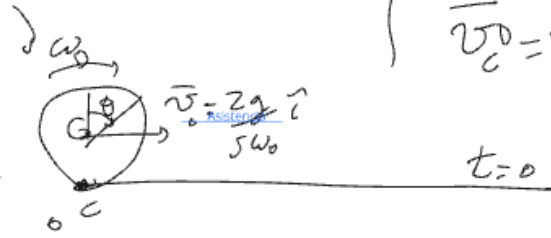
$$\widehat{r} \Delta \theta = \Delta x$$



$$\Pi \bar{a}_6 = \overline{\Pi Exr}$$

$$RSD: \boxed{\omega = \frac{\dot{x}}{r}}$$

$$\bar{v}_c = \bar{v}_6$$



$$\bar{r}_6 = x\hat{i} + y\hat{j} \longrightarrow \bar{v}_6 = \dot{x}\hat{i} + \dot{y}\hat{j} + r\dot{\theta}\hat{j} = \dot{x}\hat{i} + x\omega\hat{j} - r\omega\hat{i}$$

$$\longrightarrow \bar{a}_6 = \ddot{x}\hat{i} + \dot{x}\omega\hat{j} - x\omega^2\hat{i} - r\omega^2\hat{j} = (\dot{x} - x\omega^2)\hat{i} + (\dot{x}\omega - r\omega^2)\hat{j}$$

$\bar{z} \cdot \bar{v} \times \bar{r}$

$$\overline{\Pi Exr} = -Mg(\cos(\omega t))\hat{j} + \sin(\omega t)\hat{z} + N\hat{j} + F_r\hat{i}$$

$$\overline{\Pi Exr}^{(t=0)} = -r\hat{j} \times F_r\hat{i} = rF_r\hat{k}$$

1ª Condición:
$$\begin{cases} M(\dot{x} - x\omega^2) = F_r - Mg\sin(\omega t) \\ M(\dot{x}\omega - r\omega^2) = N - Mg\cos(\omega t) \end{cases}$$

2ª Condición:
$$M(6 - 2) \times \bar{a}_6 + \frac{d(\Pi_6 \bar{\omega})}{dt} = \overline{\Pi_6 Exr}$$

$$I_6 = \begin{pmatrix} \frac{Mr^2}{4} & 0 & 0 \\ 0 & \frac{Mr^2}{4} & 0 \\ 0 & 0 & \frac{Mr^2}{2} \end{pmatrix}$$

$$\frac{d\Pi_6 \bar{\omega}}{dt} = \overline{M Exr} = rF_r\hat{k}$$

$$\bar{\omega}_{t=0} = -\omega_0\hat{k} = \begin{pmatrix} 0 \\ 0 \\ \omega_0 \end{pmatrix}$$

$$\Pi_6 \bar{\omega} = \frac{Mr^2}{2} \omega_0\hat{k}$$

$$\frac{d\Pi_6 \bar{\omega}}{dt} = -\frac{Mr^2}{2} \dot{\omega}_0\hat{k}$$

$$\text{RSD: } \omega_0 = \frac{\dot{x}}{r}$$

$$\dot{\omega}_0 = \frac{\ddot{x}}{r}$$

$$-\frac{M r^2}{2} \dot{\omega}_0 = r F_r$$

$$-\frac{M r}{2} \ddot{x} = r F_r \rightarrow F_r = -\frac{M \ddot{x}}{2}$$

$$M(\ddot{x} - r \omega_0^2) = -\frac{M \ddot{x}}{2} - M g \sin(\omega_0 t) \rightarrow \frac{3}{2} \ddot{x} - r \omega_0^2 = -g \sin(\omega_0 t)$$

$$\textcircled{A} \quad \frac{3}{2} \ddot{x} = r \omega_0^2 \Leftrightarrow \ddot{x} = \frac{2 r \omega_0^2}{3} \rightarrow x = A e^{\sqrt{\frac{2}{3}} \omega_0 t} + B e^{-\sqrt{\frac{2}{3}} \omega_0 t}$$

$$\textcircled{P} \quad \begin{cases} x = C \sin(\omega_0 t) \\ \ddot{x} = -C \omega_0^2 \sin(\omega_0 t) \end{cases} \left\{ \begin{array}{l} -\frac{3}{2} C \omega_0^2 \sin(\omega_0 t) - \omega_0^2 C \sin(\omega_0 t) = -g \sin(\omega_0 t) \\ -C \omega_0^2 \frac{5}{2} = -g \Leftrightarrow C = \frac{2}{5} \frac{g}{\omega_0^2} \end{array} \right.$$

$$x = A e^{\sqrt{\frac{2}{3}} \omega_0 t} + B e^{-\sqrt{\frac{2}{3}} \omega_0 t} + \frac{2}{5} \frac{g}{\omega_0^2} \sin(\omega_0 t) \quad \left\{ \begin{array}{l} x(0) = 0 \\ \dot{x}(0) = \frac{2g}{5\omega_0} \end{array} \right.$$

$$x(0) = A + B = 0 \quad A = -B$$

$$\dot{x}(0) = \underbrace{\sqrt{\frac{2}{5}} \omega_0 \left(\frac{2A}{1-B} \right)}_{\geq 0} + \frac{2g}{5\omega_0} = \frac{2g}{5\omega_0} \implies A = -B = 0$$

$$\implies \boxed{x(t) = \frac{2g}{5\omega_0^2} \sin(\omega_0 t)}$$

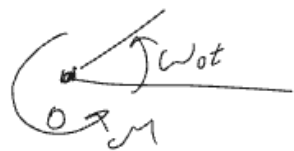
$$b) \quad \overline{F}_y = -\frac{m\ddot{x}}{2} = + \frac{mg}{5} \sin(\omega_0 t)$$

$$m(2\dot{x}\omega_0 - r\omega_0^2) = \textcircled{N} - mg \cos(\omega_0 t)$$

$$\dot{x} = \frac{2g}{5\omega_0} \cos(\omega_0 t)$$

$$\underline{N(0) > 0}$$

c) \mathcal{M}



Hay que pararse en el
sist de la barra

$$\frac{d\Pi_{\omega_0}}{dt}$$

$$= \overline{M \dot{x} r} \implies \overline{M \dot{x} r} = 0$$

$$\implies \overline{\mathcal{M} \dot{h}} + \overline{M \dot{h}_0} + \overline{M \dot{h}_0} + \overline{M \dot{h}_0} = 0$$

