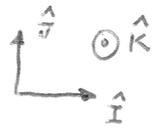
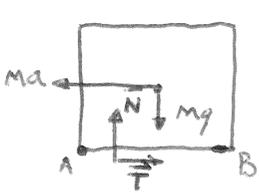


1) I) $\vec{F}_i = m_i \vec{a} \Rightarrow \vec{M}_O = \sum_i m_i (\vec{r}_i - O) \wedge \vec{F}_i = \sum_i m_i (\vec{r}_i - O) \wedge \vec{a} = (G-O) \wedge (M\vec{a})$

$$\vec{R} = \sum_i \vec{F}_i = \sum_i m_i \vec{a} = M \vec{a}$$

Entonces el sistema de fuerzas \vec{F}_i es equivalente a una única fuerza $M \cdot \vec{a}$ aplicada en el centro de masas

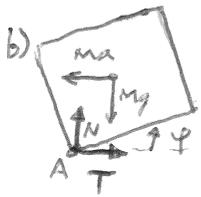
II) a) Voy a trabajar en el sistema relativo (camión), donde aparece una fuerza ficticia $\vec{F} = -m \cdot \vec{a}$ sobre el lavarropa.



1ª cardinal placa $\Rightarrow \begin{cases} T = M \cdot a \\ N = M \cdot g \end{cases}$

condición de no deslizamiento $\rightarrow T \leq f \cdot N \Rightarrow M \cdot a \leq f \cdot M \cdot g \Rightarrow \boxed{f \geq \frac{a}{g}}$

condición de no vuelco $\rightarrow \begin{cases} \vec{M}_A^{act} \cdot \hat{K} < 0 \\ \vec{M}_B^{act} \cdot \hat{K} > 0 \end{cases} \Rightarrow \begin{cases} \vec{M}_A^{act} \cdot \hat{K} = \frac{M \cdot l}{2} (a - g) < 0 \\ \vec{M}_B^{act} \cdot \hat{K} = \frac{M \cdot l}{2} (a + g) > 0 \end{cases} \Rightarrow \boxed{\frac{a}{g} < 1}$



$d = \frac{l}{\sqrt{2}} \rightarrow$ semidiagonal placa

Aplico primera cardinal en $t=0 \rightarrow M d \cdot \ddot{\varphi} \cdot \hat{e}_\varphi|_{t=0} = (T - Ma) \hat{i} + (N - Mg) \hat{j}$

$\hat{e}_\varphi|_{t=0} = \frac{\sqrt{2}}{2} (\hat{j} - \hat{i}) \Rightarrow M d \cdot \ddot{\varphi}(0) \cdot \frac{\sqrt{2}}{2} (\hat{j} - \hat{i}) = (T - Ma) \hat{i} + (N - Mg) \hat{j} \Rightarrow \begin{cases} -M \cdot \frac{l}{2} \ddot{\varphi}(0) = T - Ma \\ M \cdot \frac{l}{2} \ddot{\varphi}(0) = N - Mg \end{cases}$

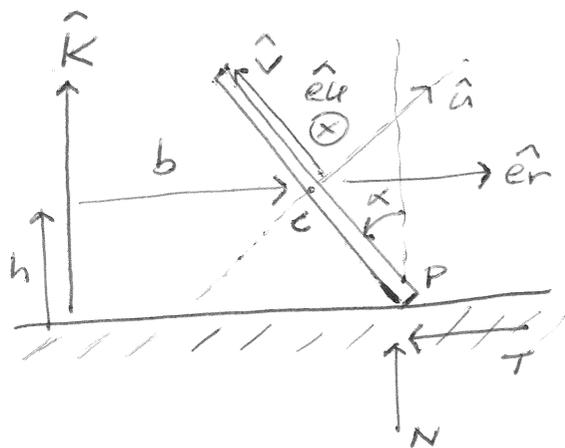
$I_A = I_G + M \cdot d^2 = \frac{M l^2}{6} + \frac{M l^2}{2} = \frac{4}{6} M l^2 = \frac{2}{3} M l^2$

Ahora aplico 2ª cardinal en A $\rightarrow \frac{2}{3} M l^2 \ddot{\varphi}(0) = Ma \cdot \frac{l}{2} - Mg \cdot \frac{l}{2} \Rightarrow \ddot{\varphi}(0) = \frac{3}{4l} (a - g)$
 proyectada en la dirección de \hat{K}

$\Rightarrow \begin{cases} N = Mg + M \frac{l}{2} \ddot{\varphi}(0) = \frac{M}{8} (5g + 3a) \\ T = Ma - M \frac{l}{2} \ddot{\varphi}(0) = \frac{M}{8} (3g + 5a) \end{cases}$

vuelco $\rightarrow \ddot{\varphi}(0) > 0 \Rightarrow \boxed{\frac{a}{g} > 1}$

no deslizamiento $\rightarrow T \leq f \cdot N \Rightarrow f \geq \frac{3g + 5a}{5g + 3a} \Rightarrow \boxed{f \geq \frac{3 + 5a/g}{5 + 3a/g}}$



$$\cos \alpha = \frac{h}{a}$$

$$\hat{u} = \cos \alpha \hat{e}_r + \sin \alpha \hat{K}$$

$$\hat{v} = \cos \alpha \hat{K} - \sin \alpha \hat{e}_r$$

$$(\hat{K} = \cos \alpha \hat{v} + \sin \alpha \hat{u})$$

$$a) \dot{\varphi} = v/b ; \quad \vec{\omega} = \dot{\varphi} \hat{K} + \dot{\varphi} \hat{u}$$

Como la moneda rueda sin deslizar! $\vec{v}_P = 0 \Rightarrow \vec{v}_C = \vec{v}_P + \vec{\omega} \times (C-P)$
 $= \vec{\omega} \times (C-P) :$

$$v \hat{e}_u = (\dot{\varphi} \hat{K} + \dot{\varphi} \hat{u}) \times a \hat{v}$$

$$v \hat{e}_u = \left[\dot{\varphi} (\cos \alpha \hat{v} + \sin \alpha \hat{u}) + \dot{\varphi} \hat{u} \right] \times a \hat{v}$$

$$= \left[\frac{v}{b} \cos \alpha \hat{v} + \left(\frac{v}{b} \sin \alpha + \dot{\varphi} \right) \hat{u} \right] \times a \hat{v} = -a \left(\frac{v}{b} \sin \alpha + \dot{\varphi} \right) \hat{e}_u$$

$$\Rightarrow \boxed{\dot{\varphi} = -\frac{v}{a} \left(\frac{a}{b} \sin \alpha + 1 \right)} ; \quad \boxed{\vec{\omega} = \frac{v}{b} \cos \alpha \hat{v} - \frac{v}{a} \hat{u}}$$

$$b) \vec{\tau}_G = -b \dot{\varphi}^2 \hat{e}_r \Rightarrow -\frac{mv^2}{b} \hat{e}_r = -T \hat{e}_r + (N - mg) \hat{K}$$

$$\Rightarrow \boxed{\begin{matrix} T = \frac{mv^2}{b} \\ N = mg \end{matrix}}$$

$$c) \vec{L}_G = \vec{M}_G^{(ext)} = \vec{M}_C^{(ext)} = (hT - a \sin \alpha N) \hat{e}_u = \left(\frac{hmv^2}{b} - a \sin \alpha mg \right) \hat{e}_u$$

$$\vec{L}_C = \mathbb{I}_C \vec{\omega} = \frac{ma^2}{4} \frac{v}{b} \cos \alpha \hat{v} - \frac{ma^2}{2} \frac{v}{a} \hat{u}$$

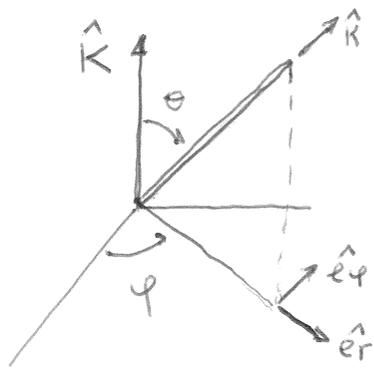
$$\dot{\vec{L}}_C = \dot{\varphi} \hat{K} \times \vec{L}_C = \left(\frac{v}{b} \right)^2 \frac{ma^2}{4} \cos \alpha \hat{K} \times \hat{v} - \sqrt{2} \frac{2}{b} \frac{m}{2} \hat{K} \times \hat{u}$$

$$= -\frac{mv^2 a}{2} \frac{1}{b} \left(\frac{1}{2} \cos \alpha \frac{a}{b} \sin \alpha + \cos \alpha \right) \hat{e}_u$$

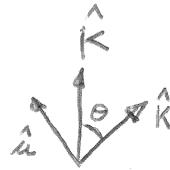
$$\Rightarrow \frac{hmv^2}{b} - mga \sin \alpha = -\frac{mv^2 a}{2} \frac{1}{b} \left(\frac{a}{2b} \sin \alpha + 1 \right) \cos \alpha \frac{h}{a}$$

$$\Rightarrow \boxed{v^2 = \frac{gab \sin \alpha}{h \left[\left(1 + \frac{a}{4b} \right) \sin \alpha + \frac{1}{2} \right]}}$$

3)



$$\hat{u} = \hat{K} \cdot \hat{e}_\varphi \Rightarrow \{\hat{e}_\varphi, \hat{u}, \hat{K}\} \text{ base de ejes principales}$$



$$\hat{K} = \cos \theta \cdot \hat{u} + \sin \theta \cdot \hat{e}_\varphi$$

a)

$$\vec{\omega} = \dot{\varphi} \hat{K} + \dot{\theta} \cdot \hat{e}_\varphi = \dot{\varphi} \cdot \sin \theta \cdot \hat{u} + \dot{\theta} \cdot \hat{e}_\varphi + \dot{\varphi} \cdot \cos \theta \cdot \hat{K}$$

$$I_{O, \{\hat{e}_\varphi, \hat{u}, \hat{K}\}} = \frac{Ml^2}{3} \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{pmatrix} \Rightarrow \vec{L}_O = \frac{Ml^2}{3} [\dot{\varphi} \cdot \sin \theta \cdot \hat{u} + \dot{\theta} \cdot \hat{e}_\varphi]$$

$$T = \frac{1}{2} \vec{\omega} \cdot I_{O, \{\hat{e}_\varphi, \hat{u}, \hat{K}\}} \cdot \vec{\omega} = \frac{Ml^2}{6} [\dot{\varphi}^2 \cdot \sin^2 \theta + \dot{\theta}^2]$$

b) * se conserva la energía porque la única fuerza ~~manuscrita~~ que trabaja es el peso y es conservativa, ya que la reacción en O, es de potencia nula porque $\vec{v}_O = 0$

* se conserva $\vec{L}_O \cdot \hat{K}$ porque la única fuerza que hace momento en O es el peso y dicho momento no tiene componente vertical.

$$c) L_z = \vec{L}_O \cdot \hat{K} = cte \Rightarrow L_z = \frac{Ml^2}{3} [\dot{\varphi} \cdot \sin \theta \cdot \underbrace{\hat{u} \cdot \hat{K}}_{\sin \theta} + \dot{\theta} \cdot \underbrace{\hat{e}_\varphi \cdot \hat{K}}_0] = \dot{\varphi} \cdot \sin^2 \theta = \underbrace{\dot{\varphi} \cos \sin^2(\pi/2)}_w$$

$$\Rightarrow \dot{\varphi} = \frac{w}{\sin^2 \theta}$$

$$E = T + U = cte \Rightarrow E = \frac{Ml^2}{6} [\dot{\varphi}^2 \cdot \sin^2 \theta + \dot{\theta}^2] + Mgl \cdot \cos \theta = \frac{Ml^2}{6} \cdot w^2$$

$$\Rightarrow \frac{Ml^2}{6} \left[\dot{\theta}^2 + \frac{w^2}{\sin^2 \theta} \right] + Mgl \cdot \cos \theta = \frac{Ml^2}{6} w^2 \Rightarrow \dot{\theta}^2 = w^2 \left[1 - \frac{1}{\sin^2 \theta} \right] - 3(g/l) \cdot \cos \theta$$

$$\Rightarrow \dot{\theta}^2 = \frac{w^2 \cdot \cos^2 \theta}{\cos^2 \theta - 1} - 3(g/l) \cdot \cos \theta$$

d) $\dot{\theta}^2 > 0 \Rightarrow \frac{w^2 \cdot \cos^2 \theta}{\cos^2 \theta - 1} - 3(g/l) \cdot \cos \theta > 0$ hago el cambio de variable $x = \cos \theta$

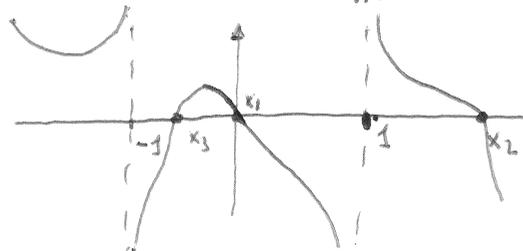
$$F(x) = \frac{w^2 \cdot x^2}{x^2 - 1} - 3(g/l) \cdot x > 0$$

ahora busca los ceros de esta función para analizar su signo

$$x \left(\frac{w^2}{x^2 - 1} - 3g/l \right) = 0 \Rightarrow x_1 = 0, \quad x^2 - \frac{3g/l \cdot w^2}{3g/l} \cdot x - 1 = 0 \Rightarrow \begin{cases} x_2 = \left(\frac{w^2}{3g/l} + \sqrt{\left(\frac{w^2}{3g/l} \right)^2 + 4} \right)^{1/2} \\ x_3 = \left(\frac{w^2}{3g/l} - \sqrt{\left(\frac{w^2}{3g/l} \right)^2 + 4} \right)^{1/2} \end{cases}$$

$$\Rightarrow x_2 = \frac{\sqrt{5} + 3}{2} = 2,62$$

$$x_3 = \frac{\sqrt{5} - 3}{2} = -0,38$$



$F(x) > 0$ PARA $x_3 \leq x \leq 0$

$$\Rightarrow 90^\circ \leq \theta \leq \arccos(-0,38)$$